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RADIATIVE AND FREE CONVECTIVE HEAT TRANSFER FROM A CONTAINERLESS SPHERE

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TECHNICAL MEMORANDUM

RADIATIVE AND FREE CONVECTIVE HEAT TRANSFER FROM A CONTAINERLESS SPHERE

INTRODUCTION

Containerless processing of copper in an electromagnetic levitation device uses the natural heating of the sample (generated by the induced eddy currents) and cooling by a helium-argon gas mixture. Such a system has been in operation in the Space Sciences Laboratory, Marshall Space Flight Center, and significant undercooling during solidification has been noted. Temperatures on the order of 2000 K were known to be reached when approximately 20 torr of argon were present in the levitation chamber, and small amounts of helium (approximately 100 torr pressure) were known to reduce the copper sphere's temperature by more than 1000 K. As the coolant gases were admitted to the evacuated chamber through a valve at a large relative distance to the sphere, heat transfer could be considered to be accomplished by radiation and free convection.

In support of this experimentation, a relatively simple mathematical model for the prediction of heat transfer from the sphere was derived. Primary conditions to be considered were the nature of heat loss, the high temperatures and low pressures involved, and the restrictions on direct experimental measurements of various parameters (sphere diameter, emissivity and temperature, and gas temperature and pressure were the only measurable quantities). Parameters to be varied in the calculation of temperature were power input to the levitation coil, emissivity and diameter of the sphere, and gas pressures.

DETERMINATION OF HEAT TRANSFER FORMULA

Power input (PI) to the sphere is equal to total heat loss through radiation and free convection:

$$PI = Q_{rad} + Q_{conv}$$

The heat transfer due to radiation (Q_{rad}) is

$$Q_{\text{rad}} = \text{orA}(T_1^A - T_0^A)$$

where

 $\sigma = \text{Stefan-Boltzman constant} = 0.56697 \times 10^{-8} \text{ W/m}^2 - \text{K}^4$

ε = emissivity of sphere

A = surface area of sphere, cm²

T₁ = sphere temperature, K

To = ambient gas temperature, K

The heat transfer due to free convection (Q_{conv}) is

$$Q_{conv} = h_c A(T_1 - T_0)$$

where h_c = heat transfer ratio for free convection [1] or, putting h_c in terms of the Nusselt number, Nu,

$$Q_e = A(T_1 - T_o) \frac{k}{D} Nu$$

where k = thermal conductivity of coolant gas and D = sphere diameter.

Yuge [2] empirically determined the Nusselt number for a sphere in air with heat loss due to free convection to be

$$Nu = 2 + 0.39 \text{ Gr}^{1/4}$$
 for $1 < Gr < 10^5$

where Gr = Grashof number. For Grashof numbers much less than one, Yuge recommended Mahony's equation:

$$Nu = 2 + O (Gr^{1/2})$$

where O = order of magnitude. By fitting a line to the lower portion of Yuge's data, O was found to equal 0.25. In his work, Mahony dealt

mainly with Grashof numbers between 10^{-3} and 10^{-9} [3]. Because the Grashof numbers encountered fell between 10^{-3} and one, both formulas were tried. No significant differences in results were found. Yuge's equation was used because of the greater documentation and certainty in the Grashof coefficient.

The Grashof number was calculated in the same manner as Yuge:

$$Gr = \frac{gD^3 \Lambda T}{T_o v_f^2}$$

where

g = acceleration due to gravity

D = sphere diameter

 ΔT = sphere temperature T_1 - gas temperature T_0

 $T_o = gas temperature; 1/T_o = heat transfer coefficient (<math>\beta$)

 v_f = kinematic viscosity at film temperature $\frac{T_1 + T_0}{2}$

$$= \frac{\rho_{\mathbf{f}}}{\mu_{\mathbf{f}}}$$

where

 ρ_f = gas density at film temperature

 $\mu_{\mathbf{f}}$ = dynamic viscosity at film temperature.

PROPERTIES OF COOLANT GASES

Because no direct measurement of gas properties (with the exceptions of pressure and temperature) was possible, methods for estimating the thermal conductivity, dynamic viscosity, and density were either derived or adapted. All properties were found to be highly temperature dependent, and the temperature difference $\rm T_1$ – $\rm T_o$ between the sphere

and the surrounding gas was very large. Therefore, all properties were evaluated at the film temperature $T_f = (T_1 - T_0)/2$, as recommended by Yuge.

In the low pressure range found in the experiments, thermal conductivity k is not pressure dependent [4]. A temperature dependency of $T_{\,\rm f}^{\,\,3/4}$ was determined empirically from data given in McAdams [1]:

$$k = k_0 (0.0173) \left(\frac{T_f}{273}\right)^{0.75}$$
 W/cm K

where

k = thermal conductivity of specific gas at 273 K in Btu/hr-ft-°F

0.0173 = conversion factor from Btu/hr-ft-oF to W/cm K.

This formula was weighted to provide more precise correlation with McAdams' data for higher temperatures and helium, the major component gas.

Since mixtures of helium and argon do not involve rotational and vibrational degrees of freedom, Brokaw's simple empirical method for calculation of thermal conductivity [5] could be used. Brokaw utilized the fact that thermal conductivities for mixtures fall between the values found by simple mixing of the component conductivities and inverse mixing of the components:

$$x_1 k_1 + x_2 k_2 > k_m > \frac{1}{\frac{x_1}{k_1} + \frac{x_2}{k_2}}$$

where x_1, x_2 = mole fractions of component gases. Brokaw represented the mixture conductivity as a combination of simple and inverse mixing:

$$k_{\rm m} = qk_{\rm SM} + (1 - q)k_{\rm RM}$$

where

$$k_{SM} = \chi_1 k_1 + \chi_2 k_2$$

and

$$\frac{1}{k_{RM}} = \frac{\chi_1}{k_1} + \frac{\chi_2}{k_2}$$

Brokaw then fit this curve to experimental data of numerous binary gas mixtures, including helium-argon, for various molar fractions to determine the value of q. The value of q was found to vary with the ratios of the gases [5]:

Brokaw's method was found to have approximately the same accuracy as other methods studied and to be much sampler in form. Therefore, Brokaw's method for gas mixtures was used in combination with the author's formula for individual gases to estimate thermal conductivities of the coolant gases.

Because the dynamic viscosities and the densities of the gases were needed in the calculation of the Grashof number, methods to calculate these properties were also needed. For the component gases, dynamic viscosity μ was taken to be

$$\mu = \mu_{o} \sqrt{\frac{T_{f}}{373}} \text{ Poise}$$

where μ_0 = dynamic viscosity of specific gas at 373 K (P), again by fitting a curve to McAdams' data [1]. A pressure dependency was not incorporated due to the high temperature and low pressure values to be used.

Brokaw's method for calculation of viscosity of gas mixtures was used, as recommended by Reid, Prausnitz, and Sherwood. Brokaw's method is based on Sutherland's approximation

$$\mu_{m} = \sum_{\substack{i=1 \\ j=1}}^{n} \frac{\chi_{i}^{\mu}_{i}}{n}$$

using

$$\phi_{ij} = \left(\frac{\mu_i}{\mu_j}\right)^{1/2} s_{ij} A_{ij}$$

where

Sii = Sutherland constant

 A_{ij} = a complicated function of molecular weights of the component gases.

 A_{ij} can be taken from a chart [6] or a graph [4] and, for mixtures of nonpolar gases, S_{ij} can be set equal to one. Thus, for mixtures containing nonpolar gases,

$$\mu_{m} = \sum_{i=1}^{n} \frac{\chi_{i} \sqrt{\mu_{i}}}{\frac{\chi_{i}}{\sqrt{\mu_{i}}} \sum_{\substack{j=1 \ j \neq 1}}^{n} \frac{A_{ij}}{\sqrt{\mu_{i}}} \chi_{j}} .$$

This mixture formula has its best results for mixtures of inert gases; overall errors for nonpolar gas mixtures ranged from 0.6 percent to 2.5 percent error [6]. Brokaw's method can also be applied at any temperature, a vital feature for this project.

The densities of the gases were calculated by the formula

$$\rho = \left(\frac{M}{22.4 \times 10^3}\right) \left(\frac{273}{T_f}\right) \left(\frac{P}{760}\right) g/cm^3$$

where T_f is in K, P is in torr, and M = molecular weight of gas. The method of simple mixing was used to calculate the mixture density:

$$\rho_{\rm m} = \chi_1 \rho_1 + \chi_2 \rho_2 \quad .$$

THE COMPUTER PROGRAM

A FORTRAN program was designed to simplify calculations and extend the range of applicability to experimentation. Appendix A presents the program nomenclature, including a cross-reference to the nomenclature used in the text of this report; Appendix B presents a program listing.

In the program, nested DO loops are used to allow for variances of sphere emissivity EP, sphere diameter D, and power input PIN (Set Experimental Parameters section) in addition to argon pressure P1. Two separate DO loops are provided to change gas pressure (Increase Pressure and Decrease Pressure sections), simulating the experimental input and pump down of gas. The Increase Prescure loop uses the incremented mole fraction to calculate helium pressure P2 for constant P1 while the Decrease Pressure loop steadily decreases the total pressure PRTTL, holding the molar ratio of the gas mixture constant. Each pressure loop includes an implied loop within which the gas properties and sphere temperature are calculated (Iterate to Satisfy Temperature Equation sections). The gas properties are determined by solving the equations presented earlier in this report; the temperature is found using an iterative technique wherein the basic heat transfer is solved for sphere temperature T by using an estimated sphere temperature T1. T, is initialized at RADT, the sphere temperature should only radiative heat transfer be present. The T1's are revised as the implied loop is repeated until the difference $|\mathbf{T} - \mathbf{T}_1|$ is less than an input tolerance (in this case, 10 K). An iteration check (FLAG, FLAGM) is also provided to check for bad convergence.

RESULTS AND CONCLUSIONS

As pressure is increased, convection takes on an increasingly important role in cooling the sphere. Once the helium enters, convection becomes dominant. Varying the amount of argon initially present does not significantly change the sphere temperature (Figs. 1 and 2). The percentage of helium present in the mixture, and not the total amount of helium, determines the sphere temperature: 90 torr of helium added to 10 torr of argon changes the sphere temperature from 1577 K to 637 K, while 360 torr helium added to 40 torr argon effects a change from 1519 K to 618 K. This phenomenon is due to the fact that the Grashof number is the only pressure-dependent term in the heat transfer equation. As expected, emissivity produced the majority of its effect at

low pressure, where Rad T is proportional to emissivity (Fig. 3). Sphere temperature is directly proportional to power input (Fig. 4) and inversely proportional to sphere diameter (Fig. 5). Once convective cooling dominates, an increase of 1.5 W in power is comparable to a decrease of 0.135 cm in diameter. During radiation's dominance, the power input change will not be as effective as the change in diameter.

Small fluctuations in the value of $k_{\rm mix}$ will effect great changes in the values of Comb T, while Rad T will remain constant. It is believed that the major source of error in this program is in the value of $k_{\rm mix}$. As temperatures increase, the estimation for individual thermal conductivities will tend to be high. This deviation will decrease the apparent pressure effect on sphere temperature Comb T.

Correlation for the *Increase Pressure* portion of the program with experimental results was found to be good: the variance between sphere temperature at 20 torr argon and 200 torr total was 2.7 percent. The actual temperature values were approximately 5 percent higher than expected; however, the error in calculation of thermal conductivity would predict a higher power input (and thus temperature) for the same cooling effect on the sphere.

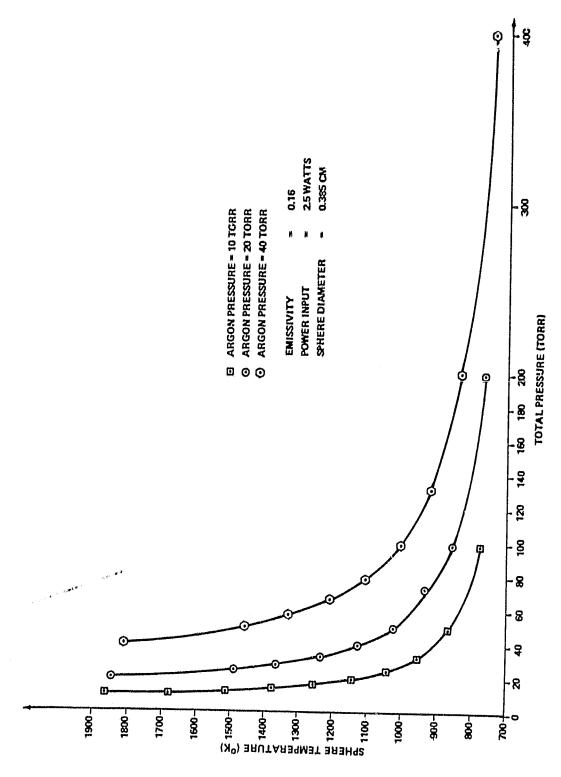
Results for the *Decrease Pressure* portion of the program were discouraging. The increase in temperature is only approximately one-third that found experimentally. This is believed to be due to the dominance of the conductivity term, which depends upon the ratio of the gases rather than the total amount of gas present (Fig. 6). The small temperature change shown illustrates the temperature dependence on the Grashof number.

NOTE TO FIGURES:

Because total heat loss did not vary significantly with the total amounts of gas present but rather with the percentage of helium present (Figs. 1,2), Figures 2 through 5 are plotted as temperature versus percentage helium. Ali lata (except where noted) were taken from runs with 20 torr argon initial pressure. The following is an approximation of results for runs beginning with 10 or 40 torr argon:

TO	tal	Pressure

He, Percent	10 torr Ar	20 torr Ar	40 torr Ar
0	10.0	20.0	40.0
10	11.1	22.2	44.4
20	12.5	25.0	50.0
30	14.3	28.6	57.1
40	16.7	33,3	66.7
50	20.0	40.0	80.0
60	25.0	50.0	100.0
70	33.3	66.7	133.3
80	50.0	100.0	200.0
90	100.0	200.0	400.0



Sample temperature as a function of total pressure for different initial argon pressures. Figure 1.

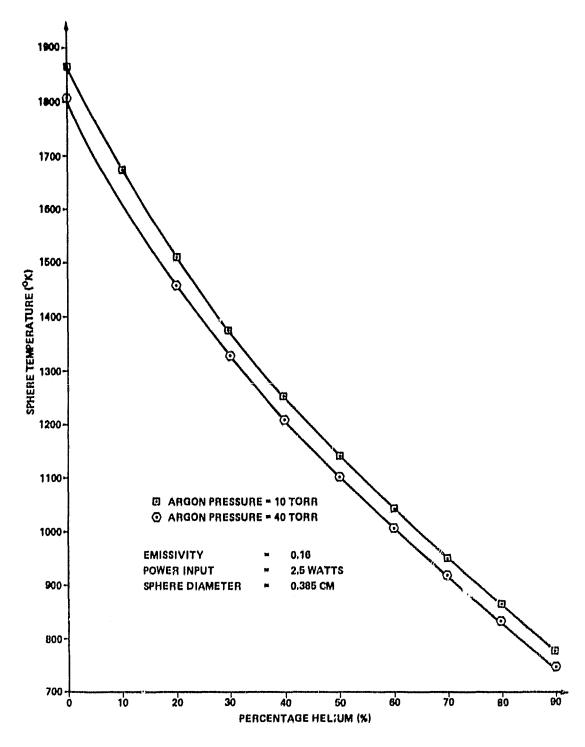


Figure 2. Sample temperature as a function of percentage helium for different initial argon pressures.

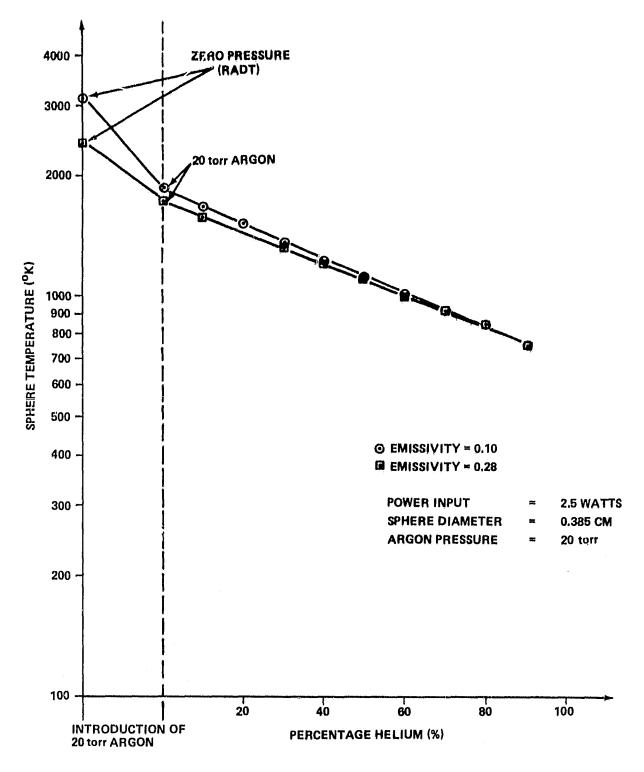


Figure 3. Sample temperature as a function of percentage helium for various emissivities.

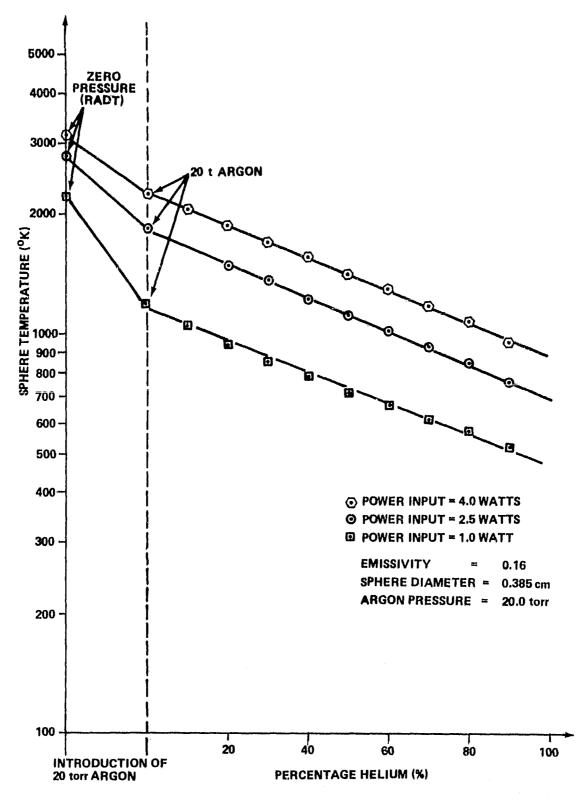


Figure 4. Sample temperature as a function of percentage helium for various power inputs.

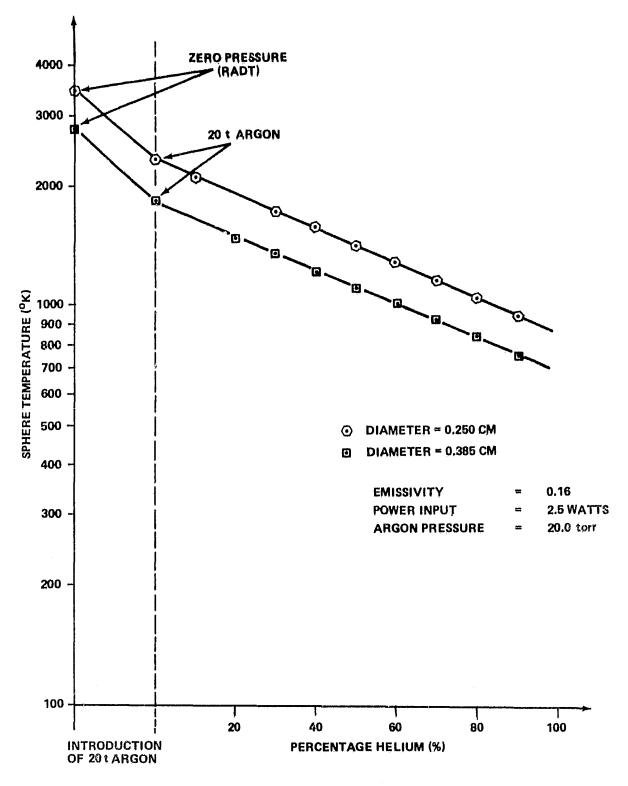
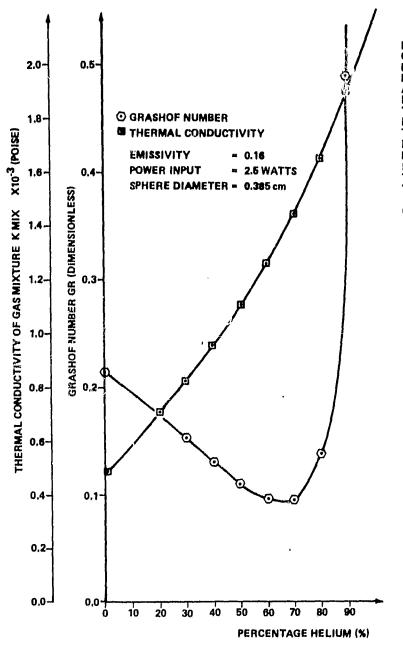


Figure 5. Sample temperature as a function of percentage helium for various sphere diameters.



NOTE: BOTH THERMAL
CONDUCTIVITY AND GRASHOF
NUMBER ARE FUNCTIONS OF
PRESSURE (EITHER TOTAL, OR
AS REPRESENTED BY PERCENTAGE
HELIUM) AND TEMPERATURE,
THUS, THE VARIANCES SHOWN
HERE ARE NOT DUE SOLELY TO
THE VARIANCE OF PERCENTAGE
HELIUM, INCREASE IN
PERCENTAGE HELIUM
IMPLIES A DECREASE IN
TEMPERATURE. THIS
TEMPERATURE DEPENDENCY
IS INCORPORATED IN THE
CURVES SHOWN.

Figure 6. Thermal conductivity and Grashof numbers as functions, of percentage helium.

REFERENCES

- 1. McAdams, W. H.: Heat Transmission. 3rd edition, McGraw-Hill Book Co., Inc., New York, 1954, pp. 165, 457, 468-469.
- 2. Yuge, T.: Experiments on Heat Transfer from Spheres Including Combined Natural and Forced Convection. Journal of Heat Transfer, vol. 82C, 1960, pp. 214-220.
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- 4. Reid, Robert C.; Prausnitz, John M.; and Sherwood, Thomas K.: The Properties of Liquids and Gases. 3rd edition, McGraw-Hill Book Co., Inc., New York, 1977, pp. 501, 416-417.
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APPENDIX A

NOMENCLATURE

Report	Program	<u>Definition</u>
A ₁₂ ,A ₂₁	A12,A21	Dimensionless parameter used in calculation of gas mixture viscosities, function of the molecu- M_1 M_2
		lar weight ratios $\left(\frac{M_1}{M_2}, \frac{M_2}{M_1}\right)$
proces -	ARGT	The argument of T used in calculation of intermediate values of sphere temperatures $T = ARGT^{1/4}$ (K^4)
-	AVDT	Absolute value of intermediate sphere temperature value difference $ T - T_1 $ (K)
Comb T	COMBT	Sphere temperature due to combined radiative and free convective heat transfer (K)
D	מ	Diameter of sphere (cm)
ε	EP	Emissivity of sphere
FLAG	FLAG	Iteration counter for temperature calculation
FLAGM	FLAGM	Maximum number of iterations allowed in calculation of sphere temperature
Gr	GR	Grashof number for gas mixture, dimensionless ratio of buoyant to inertial forces
^k o	K10,K20	Thermal conductivity of gas (1,2) at 273 K (Btu-hr-ft-°F), used in calculation of thermal conductivity of gas (W/cm-K) at film temperature
k_1, k_2	K1,K2	Thermal conductivity of gas (1,2) (W/cm-K)
k _m	KMIX	Thermal conductivity of gas mixture (W/cm-K)
M	M1,M2	Molecular weight of gas (1,2)
μ _o	MU10,MU20	Dynamic viscosity of gas (1,2) at 373 K (P) used in calculation of dynamic viscosity of gas (1,2) at film temperature (P)

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NOMENCLATURE (Concluded)

Report	Program	<u>Definition</u>
μ	MU1,MU2	Dynamic viscosity of gas (1,2) at film temperature (P)
μ_{m}	MUMIX	Dynamic viscosity of gas mixture at film temperature (P)
P_1, P_2	P1,P2	Amount of pressure of gas (1,2) (torr)
Arraines	PCNT1,PCNT2	Percentage of gas (1,2) in mixture (percent)
PI	PIN	Power input to levitation coil (W)
P _{ttl}	PTTL	Total gas pressure during increase of pressure (torr)
PRTTL	PRTTL	Total gas pressure during pump down (torr)
q	ନ୍(I)	Dimensionless parameter used in calculation of thermal conductivity of gas mixture, function of molar fraction of light gas
Rad T	RADT	Sphere temperature when only radiative cooling effects are considered (K)
ρ_1, ρ_2	RH01,RH02	Density of gas (1,2)
$\rho_{\mathbf{m}}$	RHOMX	Density of gas mixture
T	T	Intermediate sphere temperature value used in determination of sphere temperature (K)
To	TO	Ambient gas temperature (K)
T ₁	T 1	Intermediate sphere temperature value used in determination of sphere temperature (K)
$\mathbf{T}_{\mathbf{f}}$		Film temperature $\frac{T_1 + T_0}{2}$
_	TOL	Tolerance between intermediate and final sphere temperature values (K)
x_1,x_2	X1,X2	Mole fraction of gas (1,2)

APPENDIX B

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             DOUBLE PRECISION TI, RADT, T, ARGT, GR, RHOI, RHO2, RHOHX
· 028
             DOUBLE PRECISION KI, K2, KMIX, MUI, MU2, MUMIX
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9930
             DATA Q 7.32, 34, 37, 39, 42, 46, 50, 55, 61, 69/
             DATA M1740. 2, M274. 2, MU107. 000267, MU207. 000227, T07300. 7
F#31
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             WRITE(6,30) M1,M2,MU10,MU20,K10,K20
             FORMAT(37X,"1",16X,"2"/35X,"ARGON",12X,"HELIUM"/" MASS",31X,F3.6,
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            IRM COND (KO)", 11X, F5.3, 12X, F5.3///)
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1 IN MIXTURE"/" PCNT2 = PERCENTAGE OF GAS 2 IN MIXTURE"/" RADT =
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4.45
            SPHERE TEMP DUE TO RADIATIVE HEAT TRANSFER (P = 0, DEG KELVIN)"/"
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            OCMBT = SPHERE TEMP DUE TO COMBINED RADIATIVE AND CONVECTIVE HEAT THANSFER ODE KELVIN)"/" GR = GRASHOF NUMBER FOR GAS MIXTURE"
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× 163
11 64
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4472
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             P2 = X2*(P1/X1)
44. B. 3
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       b O
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11)90
             RHO2 # (3.21E-5+M2+P2)/(T1+T0)
4444
             RHOMX = X1*RHO1+X2*RHO2
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             MU2 = 3 66E+2+MU20+(T1+T0)++ 5
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             K1 = 1 53E-4*K10*((T1+T0)** 75)
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             K2 = 1 53E-4*K20*((T1+T0)**.75)
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             KHIX = Q(I)*(X1*K1+X2*K2)+(1.-Q(I))*(1./((X1/K1)+(X2/K2)))
             IF (%2+.3) 65,65,93
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29:4
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n 101
                  *T1-T0)**1 25)/(T0**,25*HUMIX**,5)))/(1 78E-12*EP*D**2))
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 4.35
             GO TO 96
35 1 3 A
       47
             T=x(PIH-(1 78E-12*EP*D*+2*(T1**4-T0**4))-(16 86*KMIX*D**1.75*
6110
             : FHONX++.5+(T1-T0)++1 25)/(T0++.25+HUMIX++.5)))/(6 28+KMIX+D))
44.5 %
6117
             AVOT # CABS(T-T1)
       96.
W 1 1 3
             1F(TOL-AVOT) 100,160,160
1114
       100
             1F FLAGM-FLAG 1 140,140,110
             11 a (T+T1)/2
0115
       1 1 n
             JF: T1-T0: 120,120,130
1 1 1 1
9417
       120
             T1 # T0 + 100
```

1 1 1 2

; **"** (i

60 TO 60

```
C CALCULATE AND WRITE FINAL DATA
11:24
9121
            WRITE (6.150) T.TI.RADT
0122
      :40
            FORMATO," BAD CONVERGENCE THE LAST
" AND ",FG O," RADY # ",F6.0/)
0123
                                          THE LAST TWO VALUES OF T WERE ".FO O.
      150
0124
0175
            GO TO 180
            PTTL # P1 + P2
0120
      160
0127
            PCHT2 * X2*100
            GR * (980 +D++3+RHONX++2+(T1-T0))/(T0+MUH1X++2)
0120
            WRITE(6,170) EP, PIN, D, PTTL, P1, P2, PCHT2, RADT, T, GR, KMIX
0129
            FORMAT(F5.2,F11.1,F11.3,F13.1,3F12.1,F13.0,F12.0,E15.3,E13.3)
0130
      170
0.131
      160
            CONTINUE
0132
0133
0134
      C DECREAGE PRESSURE
0135
0136
      C
            URITE (6,185)
0137
            FORMAT(//25X, ***********************************
0138
      135
           0139
           !*PTTL*,8X,*PRES!*,7X,*PRES2*,7X,*PCHT2*,7X,*RADT*,8X,*COMBT*,10X,
0140
           1 " GR " . [ OX , "KHIX"/ )
0141
0142
      C SET CONSTANTS AND INITIALIZE VARIABLES
0143
0144
9:45
            X1 ×
0146
            X2 # 9
0147
            PCHT2 = X2 + 100
            PRITL * PTTL
0149
            T1 = RADT
0149
            DO 280 H#1,13
0150
0151
            PRESI = X1*PRTTL
            PRES2 * X2+PRTTL
0152
            FLAG # 0
0153
0154
      ¢
      C ITERATE TO SATISFY TEMPERATURE EQUATION
0155
0156
0157
      190
            FLAG = FLAG + 1.
            RHO1 = (3.21E-5+H1+PRES1)/(T1+T0)
0158
            RHG2 = (3.21E-5+M2+PRES2)/(T1+T0)
0159
            RHONX = X1+RHO1+X2+RHO2
0160
0161
            HU1 # 3.66E-2+HU10+(T1+T0)++,5
            9U2 = 3.66E-2 + MU20 + (T1+T0) ++ .5
0162
0163
            HUHIX = ((X1+HU1)/(X1+A12+(HU1/HU2)++.5+X2))+((X2+HU2)/(X2+A21+
                     (HU2/HU1)++.5+X1))
0164
            K1 = 1.53E-4*K10*((T1+T0)**,75)
0165
            K2 = 1.53E-4+K20+((T1+T0)++,75)
0166
            KMIX = Q(I)+(X1+K1+X2+K2)+(1,-Q(I))+(1./((X1/K1)+(X2/K2)))
9167
             T = ((PIN-(1,7BE-12*EP*D**2*(T1**4-T0**4))-((6,86*KM1X*D**1,75*
0168
                 RHOMX++.5+(T1~T0)++1.25>/(T0++.25+HUMIX++.5>))/(6.28+KM1X+D))
0169
0170
                 +10
            AVPT = DARS(T-T1)
0171
             14(101-AVD1) 200,260,260
0172
0173
     500
             IF/FLAGM-FLAG) 240,240,210
            11 # (1+11)/2
      210
0174
0175
             IF(||1-||4|) 224,220,230
      220
             71 m TO + 100.
0176
0177
      530
             CO TO 190
0178
      Ĉ
```

```
0174 F CALCULATE AND WRITE FINAL DATA
0180
      C
0181
      240
            WRITE(6:250) T.YI
0182
            FORMATON BAD CONVERGENCE
      250
                                         T = *,F6.0,* T1 = *,F6 0/)
            CO TO 280

:R = (980 +D++3+RHOHX++2+(T1-T0))/(T0+MUH1X++2)
0187
0184
      260
0185
            VALIE (6.270) EP. PIH. D. PRTTL. PRESI, PRES2, PCHT2, RADT. T. GR. KMIX
            FORMATCES 2, F11. 1, F11. 3, F13 1, 3 F12. 1, F13. 0, F12. 0, E15 3, E13. 3)
0186
      2:0
0137
0188
      C CHECK IF PUNP DOWN IS COMPLETED
9189
            PRITE = PRITE-30. IF(PRITE LE.O.) GO TO 285
0190
0191
0192
      280
            CONTINUE
0193
      284
            WRITE(6.790)
0194
            590
0195
0196
      300
            CONTINUE
0197
      310
            CONTINUE
0198
      320
            CONTINUE
0199
            WRITE(6:330)
0200
      330
            FORMATCIHIS
0201
      340
            CONTINUE
0202
            STOP
11203
            EHD
0204
            EHD $
```

APPROVAL

RADIATIVE AND FREE CONVECTIVE HEAT TRANSFER FROM A CONTAINERLESS SPHERE

By Karen Johnson

The information in this report has been reviewed for technical content. Review of any information concerning Department of Defense or nuclear energy activities or programs has been made by the MSFC Security Classification Officer This report, in its entirety, has been determined to be unclassified.

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